

# Efficiency improvement of a bearingless motor with powder iron core

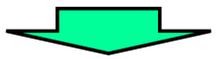


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## Introduction

### Bearingless motor

- A motor which combines magnetic bearing function and rotation in the same unit



◎ No wear particles, Lubricant-free

- Application: Chemical pumps, Blood pumps, etc

### Previous research

- Proposed a compact bearingless motor (fig.1)
- Succeeded in stable suspension and rotation
- Problem: Motor loss is large ⇒ Low efficiency (fig.3)

### Research objective

- Improvement of efficiency with powder iron core

## Proposed bearingless motor

### Structure

- Rotor: Two iron disks (with PM) and a PM ring
- Stator: C-shape ⇒ Reduction motor height
- Two kind of windings

: Suspension and motor are combined

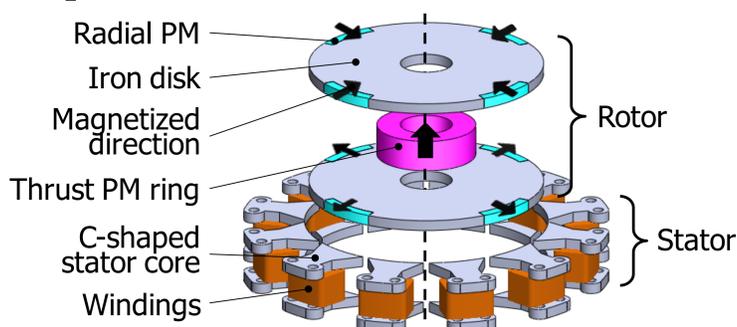


Fig.1 Proposed structure

### Magnet suspension principle

- Radial direction: Suspension force is generated by superposition of PM flux and suspension flux
- Radial position (suspension force) is actively regulated
- Axial/tilting direction: Passively stable with magnetic attractive force

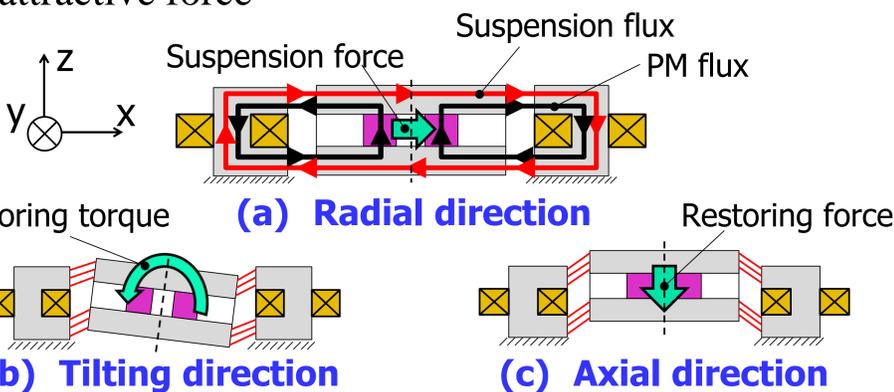


fig.2 Suspension principle

## Test machine loss (with bulk core)

### Power consumption with no load

- Motor loss is 70% or more of total loss
- Iron loss is 60% of motor loss ⇒ Eddy current

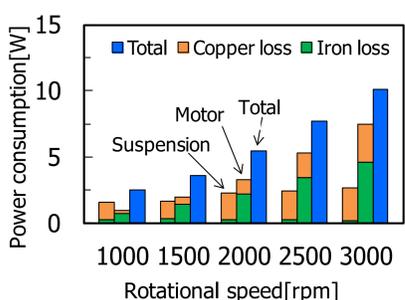


fig.3 Power consumption

$$P_{total} = P_{output} + P_{mech} + P_{copper} + P_{iron}$$

No load Frictionless

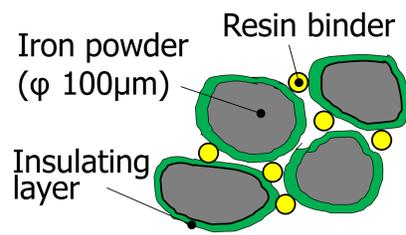
$$P_{iron} = P_{hysteresis} + P_{eddy\ current}$$

Increase

## Iron loss reduction

### Powder iron core

- Powder iron core is material which compresses insulated iron powder
- High electrical resistance ⇒ Reduced eddy current ⇒ Reduced iron loss

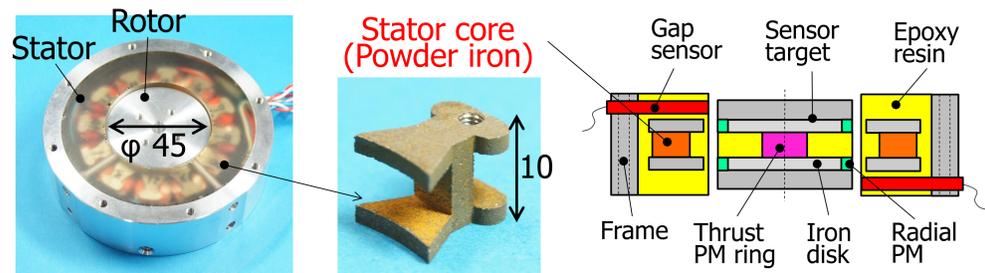


× Low permeability than bulk ⇒ Torque and force are reduced? ⇒ Investigation of motor performance with powder iron core

fig.4 Powder iron core

### Test machine with powder iron core

- Use powder iron core for stator core of proposed bearingless motor
- Stator core is molded with epoxy resin ⇒ Because powder iron core is very brittle
- Rotor motion range in radial direction is ±0.25mm



(a) Real view

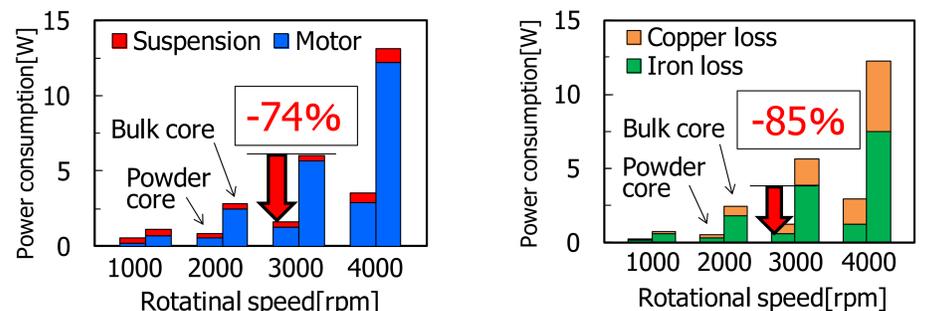
(b) Schematic view

fig.5 Fabricated test machine

## Powder iron core performance

### Power consumption with no load

- Total: 74% reduction (Motor loss reduction is large)
- Iron loss: 85% reduction ⇒ Eddy current is reduced



(a) Total

(b) Only motor

fig.6 Measured power consumption

### Torque and Efficiency

- Torque ( $k_T$ [mNm/A]) at 3000rpm: Powder > Bulk ⇒ Eddy current works as brakes
- Efficiency: 25% improvement (at 3000rpm)

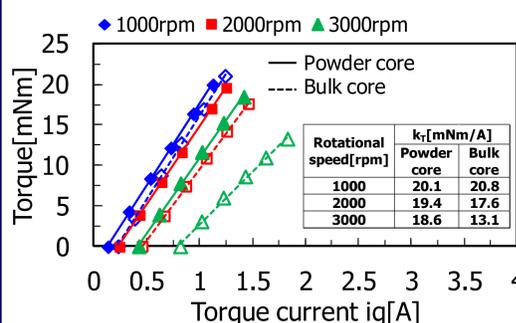


fig.7 Measured torque

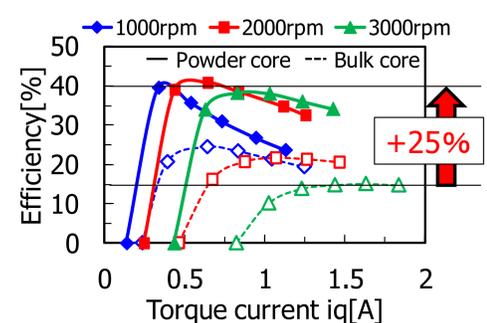


fig.8 Efficiency

## Conclusion

- It is effective to use powder iron core for proposed bearingless motor to improve efficiency

# Force Interference Reduction with Winding Configuration in a Homo-polar Bearingless Motor

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## BACKGROUND AND OBJECTIVE

### FINAL GOAL OF RESEARCH

Realization of **precise positioning** bearingless motor

Application possibility: Micro endmill cutting

Problem of conventional cutting { axial runout  
 Heterogeneous cutting force

➔ Bearingless motor

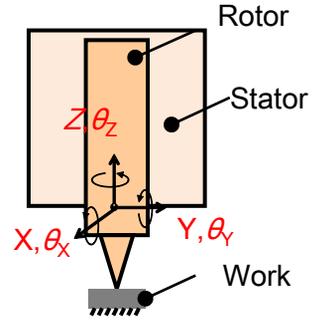
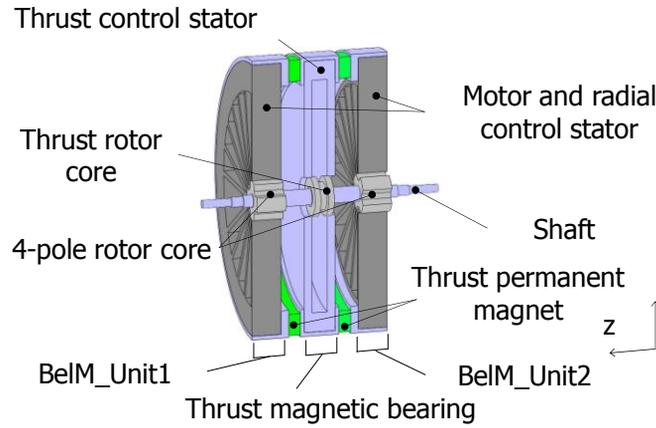


Fig.1 Concept of application

Position of endmill } Measurement and control ➔ Realization of Precision cutting  
 Cutting force }

### PROPOSED STRUCTURE



❑ 8-pole homo-polar bearingless motor  
 Permanent magnet-free (on rotor)

➔ Realization of Robust rotor

❑ 5 DOF control  
 { BelM\_Unit1 and BelM\_Unit2: X, Y, theta\_x, theta\_y  
 Thrust control bearing(TMB): Z  
 ❑ Distributed winding } Sine wave MMF  
 ❑ 24 slots

Fig.2 Proposed structure of homo-polar bearingless motor

## PRINCIPLE OF RADIAL SUSPENSION FORCE AND DIRECTION ERROR

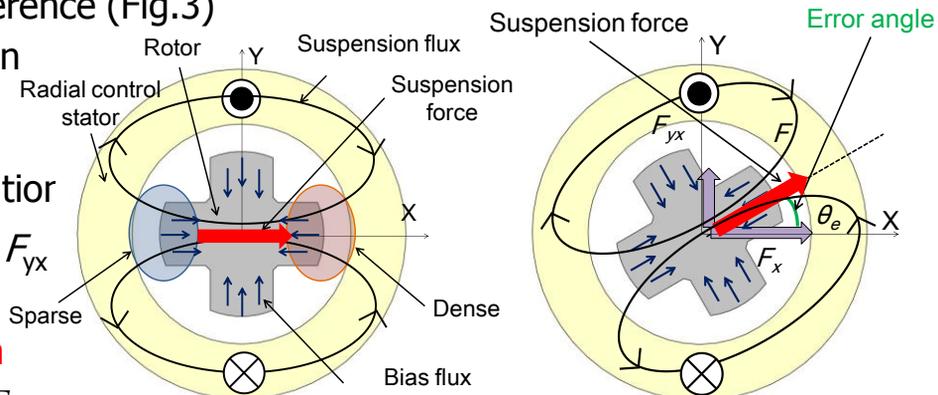
❑ Undesirable Force Interference (Fig.3)

(a) At a symmetrical position  
 Suspension force  $F_x$

(b) At an asymmetrical position  
 Suspension force  $F_x$  and  $F_{yx}$

➔ Affects the control system

Error angle  $\theta_e = \tan^{-1}\left(\frac{F_{yx}}{F_x}\right)$



(a) Symmetrical

(b) Asymmetrical

Fig.3 Principle of radial suspension force

### OBJECTIVE OF THIS PRESENTATION

Investigation of the optimal two-pole suspension winding configuration for miniaturization of an error angle (Fig.3).

## THEORETICAL CALCULATION

Calculated error angle  $\theta_e$

Calculated magnetomotive force distribution  $A(\phi) \rightarrow F_x, F_{yx}, \theta_e$

$$F_x = \frac{\mu_0 R l}{2g_0^2} \int_{-\pi}^{\pi} A^2(\phi) \cos(\phi) d\phi$$

$$F_{yx} = \frac{\mu_0 R l}{2g_0^2} \int_{-\pi}^{\pi} A^2(\phi) \sin(\phi) d\phi$$

$\mu_0$ : the permeability of free space  
 $g_0$ : the air gap length  
 $R$ : the mean air-gap radius  
 $l$ : the stack length

Single-layer winding model

Slot No.2,3: 1 turn

No.1,4:  $M$  turn

( $M=0, 0.1, 0.2, \dots, 0.9, 1$ )

U-phase 1 A  
 V-phase -0.5 A  
 W-phase -0.5 A

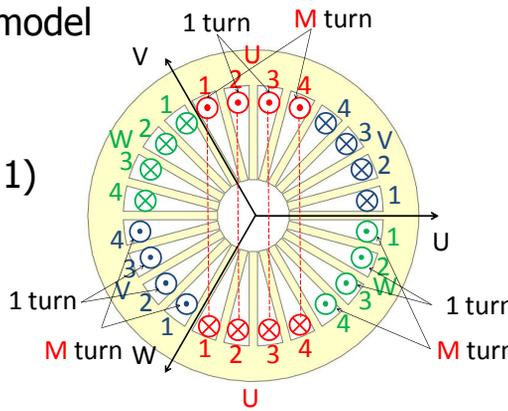


Fig.4 Single-layer winding (Model 1)

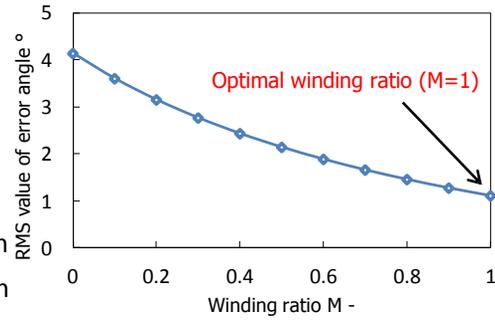


Fig.5 RMS value of error angle

Double-layer winding model (Model 2)

Slot No.1~4: 1 turn

No.5,6:  $K$  turn

No.7,8:  $L$  turn

( $K,L=0, 0.1, 0.2, \dots, 0.9, 1$ )

U-phase 1 A  
 V-phase -0.5 A  
 W-phase -0.5 A

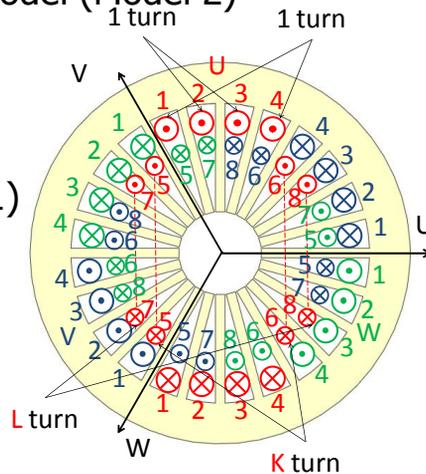


Fig.6 Double-layer winding (Model 2)

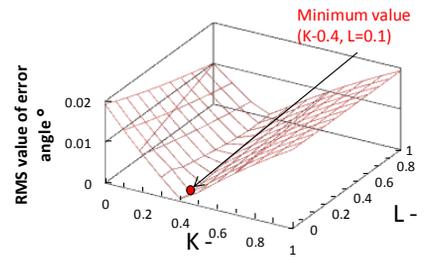


Fig.7 RMS value of error angle

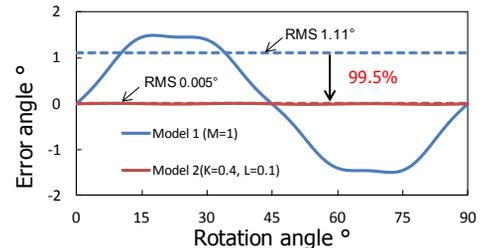


Fig.8 Comparison of Model 1 and Model 2

➔ RMS value of Model 2: 95.5% decreased against RMS value of Model 1

## 3D-FEM ANALYSIS

Analyze error angle  $\theta_e$



Certified validity of theoretical calculation

Table.1 RMS value of Theoretical calculation and FEM analysis

	Calculation	FEM
Model 1	1.11°	2.22°
Model 2	0.005°	0.300°

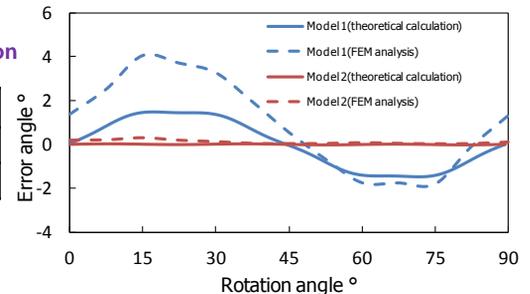


Fig.9 Comparison of theoretical calculation and FEM analysis

## CONCLUSIONS

- ❑ Suspension winding configuration of Model 2 is optimal.
- ❑ 3D-FEM analysis results agree well with theoretical calculated results.
- ❑ Production test model in near future.

# Design and Test Results of a One-DOF Actively Positioned Bearingless Motor

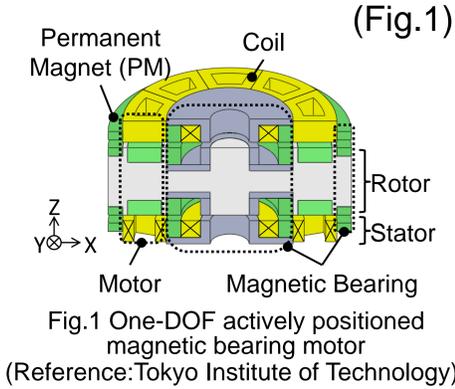
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## Introduction

### One-DOF actively positioned magnetic bearing motor

- Position control : Z direction  
 ➔ **One-DOF actively positioned**
- Required number of inverter
  - One single-phase inverter for magnetic bearing
  - One three-phase inverter for motor
- ➔ **Large size**



Magnetic bearing + Motor

### Study objective

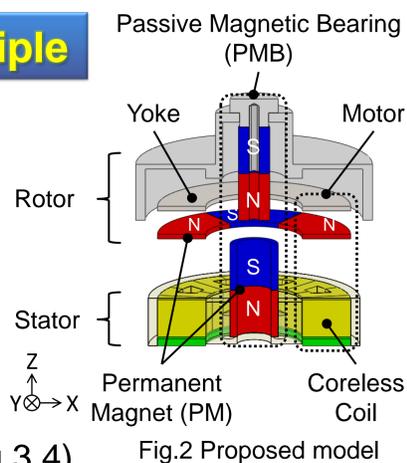
- Realize of one-DOF actively positioned **bearingless motor** only one three-phase inverter.

## Configuration & Drive Principle

### Configuration (Fig.2)

- One three-phase inverter  
 ➔ **Small size**
- 6-pole, 9-slot
- Coreless motor

Application Cooling fan



### Position control & Rotation (Fig.3,4)

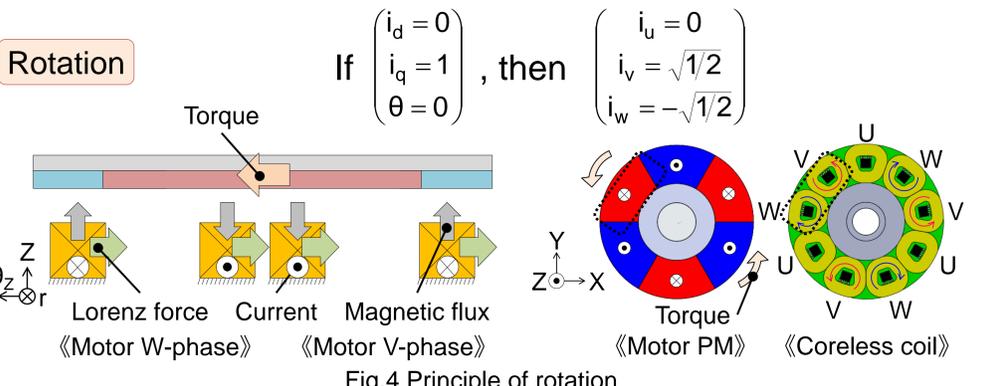
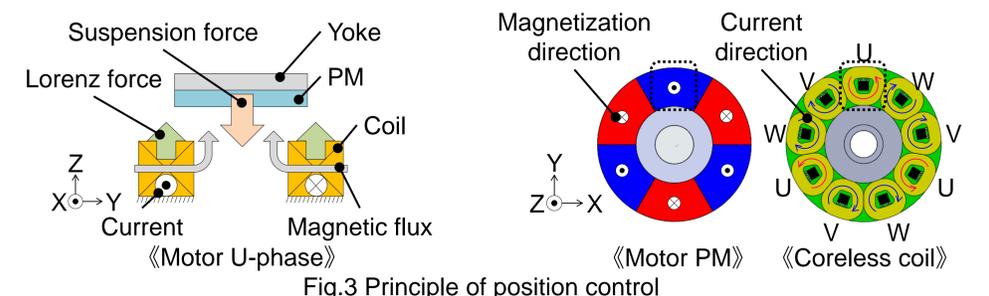
- Position control(Z) : **d-axis current** ( $i_d$ )
- Rotation( $\theta_z$ ) : **q-axis current** ( $i_q$ )

Transformation formula

$$\begin{bmatrix} i_u \\ i_v \\ i_w \end{bmatrix} = \sqrt{\frac{2}{3}} \begin{bmatrix} 1 & 0 \\ -1/2 & \sqrt{3}/2 \\ -1/2 & -\sqrt{3}/2 \end{bmatrix} \begin{bmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{bmatrix} \begin{bmatrix} i_d \\ i_q \end{bmatrix}$$

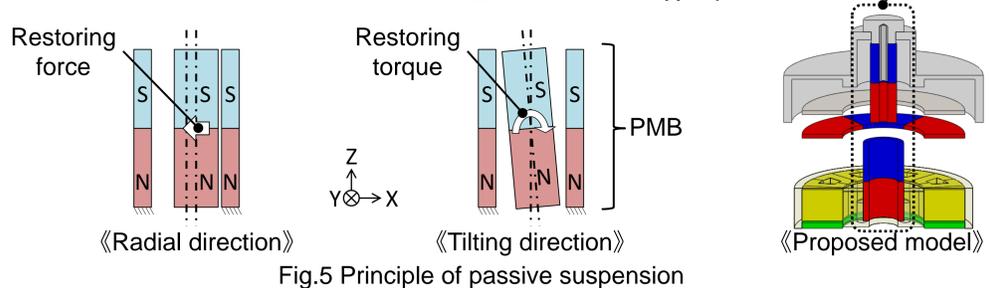
Position control

If  $\begin{pmatrix} i_d = 1 \\ i_q = 0 \\ \theta = 0 \end{pmatrix}$ , then  $\begin{pmatrix} i_u = \sqrt{2/3} \\ i_v = -\sqrt{1/6} \\ i_w = -\sqrt{1/6} \end{pmatrix}$



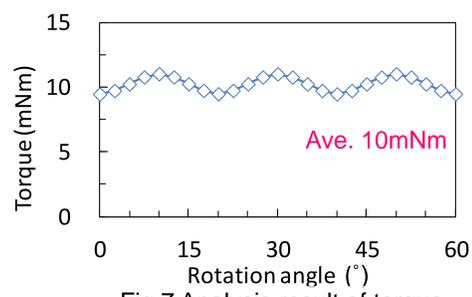
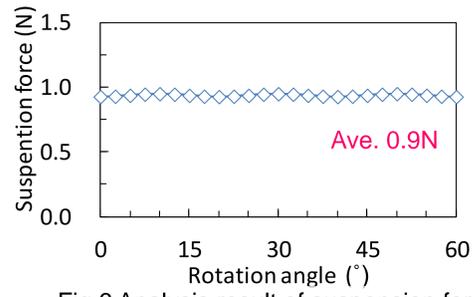
### Passive suspension (Fig.5)

- Radial direction(X,Y), Tilting direction( $\theta_x, \theta_y$ )



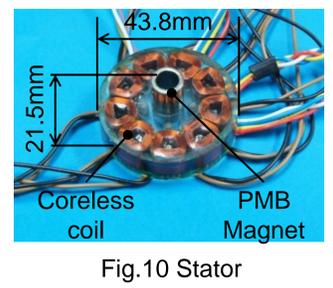
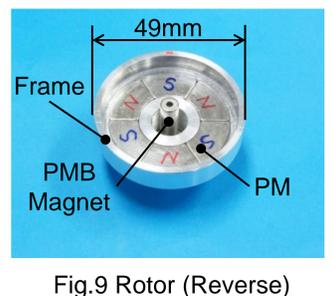
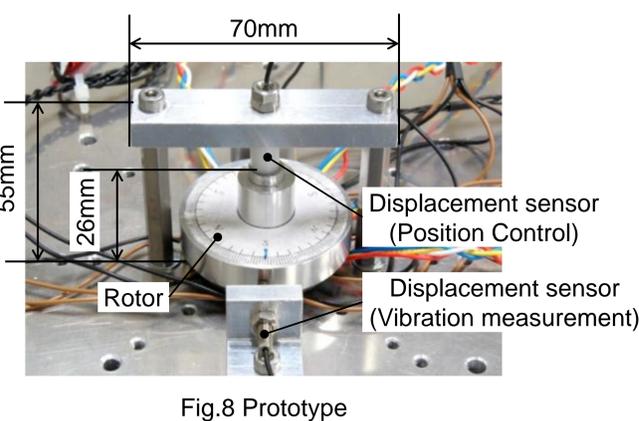
## Electromagnetic Analysis

- Rated current : 0.25A
- Rated suspension force : **0.9N** (Fig.6)
- Rated torque : **10mNm** (Fig.7)



## Test Model

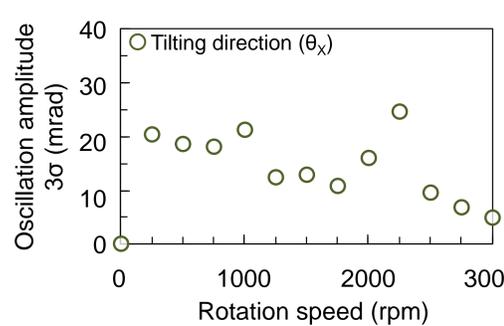
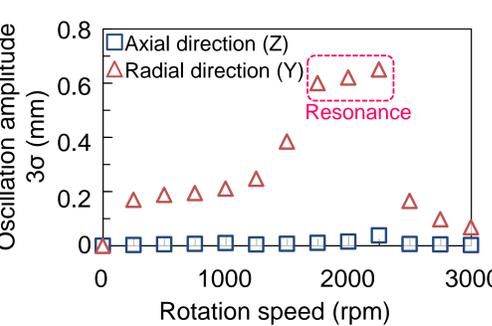
- Axial length : 26mm (Fig.8)
- Rotor diameter : 49mm (Fig.9)
- Magnetic gap : 1.0mm



## Drive test (No load)

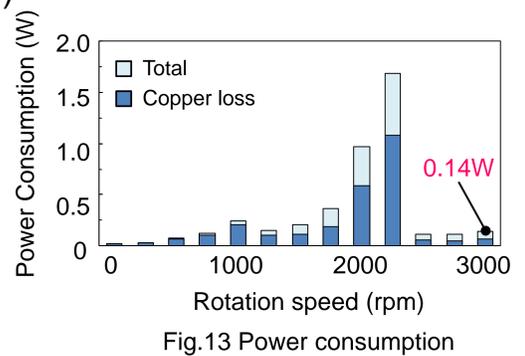
### Oscillation property (Fig.11,12)

- 2250rpm : Resonance of radial direction
- 3000rpm : Axial 4μm Tilting 4.9mrad  
 Radial 0.07mm



### Power consumption (Fig.13)

- 2250rpm Increase of oscillation  
 ➔ Friction loss Increase  
 ➔ Copper loss Increase
- 3000rpm : 0.14W



## Conclusions

- One-DOF actively positioned bearingless motor for cooling fan is proposed.
- Rated suspension force is 0.9N, and rated torque is 10mNm.
- Axial, radial and tilting oscillation amplitude at 3000rpm are 4μm, 0.07mm, and 4.9mrad.

